

# Convection enhances mixing in the Southern Ocean

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## Key Points:

- We use a convection-resolving model with wind- and buoyancy-forcing to simulate an idealised Southern Ocean domain.
- Results show that local mixing efficiency has significant spatial variability.
- Turbulent convection enhances the average mixing efficiency in the Southern Ocean such that it is greater than previously assumed.

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## Abstract

Mixing efficiency is a measure of the energy lost to mixing compared to that lost to viscous dissipation. In a turbulent stratified fluid the mixing efficiency is often assumed constant at  $\eta = 0.2$ , whereas with convection it takes values closer to 1. The value of mixing efficiency when both stratified shear flow and buoyancy-driven convection are active remains uncertain. We use a series of numerical simulations to determine the mixing efficiency in an idealised Southern Ocean model. The model is energetically closed and fully resolves convection and turbulence such that mixing efficiency can be diagnosed. Mixing efficiency decreases with increasing wind stress, but are enhanced by turbulent convection and by large thermal gradients in regions with a strongly stratified thermocline. Using scaling theory and the model results, we predict an overall mixing efficiency for the Southern Ocean that is significantly greater than 0.2, while emphasising that mixing efficiency is not constant.

## 1 Introduction

The Southern Ocean is characterised by a meridional surface buoyancy gradient and persistent westerly winds, both of which drive circulation in this region. The surface buoyancy fluxes (both cooling and salinification) lead to the formation of dense water on the Antarctic continental shelf, which is exported as a plume of downwelling water along the continental slope and mixes to become Antarctic Bottom Water [Kuhlbrodt *et al.*, 2007; Marshall and Speer, 2012; Morrison *et al.*, 2015]. To balance this loss of buoyancy and maintain the stratification, turbulent diapycnal mixing transports buoyancy downwards [Munk and Wunsch, 1998; Ledwell *et al.*, 2000]. This turbulence is at least in part energised by surface wind stress, leading to both mixing and viscous dissipation of energy in the ocean [Wunsch and Ferrari, 2004]. The contribution of mixing to the total energy loss (by mixing and dissipation) is known as the mixing efficiency [Peltier and Caulfield, 2003]. Mixing efficiency is a fundamental quantity used to calculate the rate of overturning circulation in simple theories [Munk and Wunsch, 1998], interpret observations of turbulence [Gregg, 1989; Alford *et al.*, 1999], understand the ocean energy budget [Wunsch and Ferrari, 2004] and parameterise unresolved dynamics in ocean models [Shakespeare and Hogg, 2012].

The mixing efficiency traditionally used in oceanographic calculations has been  $\eta \approx 0.2$ . This value originates from experimental estimates for mixing efficiency in stratified turbulence, where mixing efficiency has a theoretical upper bound [Linden, 1979; Osborn, 1980]. However, mixing efficiency may not be a universal constant in stratified shear flows [Peltier and Caulfield, 2003; Ivey *et al.*, 2008]. Shear flow mixing efficiencies can be smaller than 0.2, (eg. Prastowo *et al.* [2008], Caulfield and Peltier [2000], Strang and Fernando [2001]), and the value may depend on the buoyancy Reynolds number ( $Re_b$ ) and gradient Richardson number ( $Ri_g$ ) [Bouffard and Boegman, 2013; Mater and Venayagamoorthy, 2014; Salehipour *et al.*, 2016]. Higher mixing efficiencies occur in stratified flows with the presence of turbulent convection. Mixing efficiencies of 0.75 in the case of Rayleigh-Taylor instability [Davies Wykes and Dalziel, 2014], 0.5 in the case of Rayleigh-Bénard convection [Hughes *et al.*, 2013; Gayen *et al.*, 2013a], 0.6 in the case of convectively driven instability due to internal waves [Chalamalla and Sarkar, 2015] and approaching 1 in the case of horizontal convection [Gayen *et al.*, 2013b, 2014; Vreugdenhil *et al.*, 2016] have been found. In addition, the presence of rough topography could enhance mixing efficiency in the abyssal ocean [Mashayek *et al.*, 2017]. It appears likely that mixing efficiency in the ocean varies substantially as a consequence of the different roles of stratification, convection, shear and wind work from place to place. The effect of turbulent convection on mixing efficiency in the context of the Southern Ocean has not been widely considered.

61 Numerical studies into the value of mixing efficiency for the global ocean are im-  
 62 peded by the inability of ocean models to fully resolve turbulent convection. High-resolution  
 63 simulations which fully resolve turbulent mixing and convection have focussed on circula-  
 64 tion regimes driven by buoyancy forcing alone, both without rotation [Scotti and White,  
 65 2011; Gayen *et al.*, 2013b, 2014] and with rotation [Vreugdenhil *et al.*, 2016], and leave  
 66 open the question of how the results apply to the ocean. In particular, no previous simula-  
 67 tions have resolved fully turbulent convection in concert with wind stress forcing.

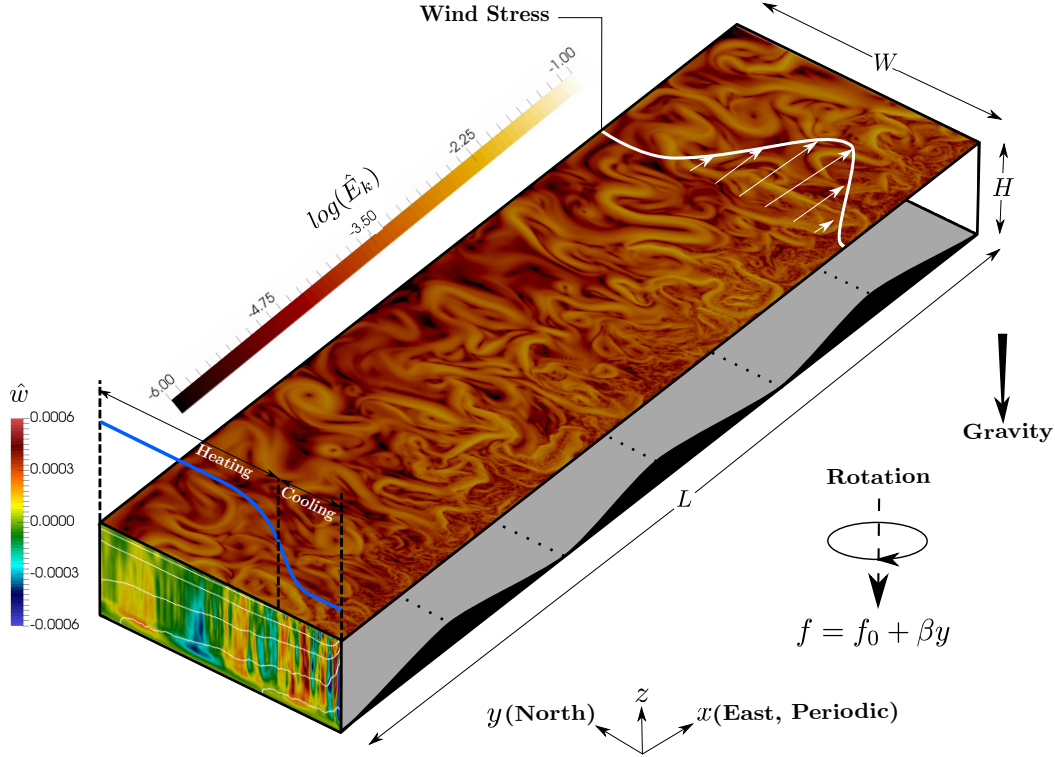
68 In this paper we use a convection-resolving model of Southern Ocean circulation  
 69 with both wind- and buoyancy-forcing to examine the local and volume-integrated diapy-  
 70 cnal mixing, dissipation and mixing efficiency. A scaling theory is developed to predict  
 71 diapycnal mixing and dissipation, and subsequently to estimate mixing efficiency in the  
 72 Southern Ocean.

## 73 2 Methodology

### 74 2.1 Model Setup

75 An idealised re-entrant zonal channel with dimensions  $H \times L \times W$  is used to represent  
 76 the circumpolar Southern Ocean (figure 1). The channel is oriented such that  $x$  is East-  
 77 ward,  $y$  is Northward, and  $z$  is vertically upward. The Coriolis parameter is  $f(y) = f_0 + \beta y$ ,  
 78 where  $\beta = df/dy$  and  $f_0 = f(y = 0)$ . The simulation is forced with a hyperbolic tangent  
 79 meridional distribution of surface temperature with an overall temperature difference  $\Delta T$   
 80 centered about the latitude  $y = W/6$  (where  $T = \Delta T/2$ ). A steady zonal wind stress is im-  
 81 posed at the surface, with meridional profile  $\tau(y) = \tau_{max} \sin^2(4\pi y/3W)$ , where  $\tau_{max}$  is the  
 82 maximal zonal wind stress and occurs at  $y = 0.375W$  from the southern boundary. The  
 83 sinusoidal lower boundary represents three Gaussian ridges (with height  $h/H = 0.2$  and  
 84 bathymetric contours aligned in the meridional  $y$  direction) as an idealised representation  
 85 of bottom topography.

93 A Direct Numerical Simulation (DNS) is used to solve the incompressible, nonhy-  
 94 drostatic Navier-Stokes (NS) momentum equations in a rotating reference frame with a  
 95 Boussinesq approximation, linear equation of state and conservation of mass and heat (see  
 96 Vreugdenhil *et al.* [2016]). Planetary vorticity is assumed to be a linear function of lat-  
 97 itude (the ‘Beta-plane’). A detailed description of the NS solver and algorithm can be  
 98 found in Gayen and Sarkar [2011] and Gayen *et al.* [2014]. The DNS are conducted us-  
 99 ing a  $1024 \times 513 \times 257$  grid that is clustered to resolve the top and bottom Ekman layers  
 100 and Stewartson boundary layers, and is uniform in the  $x$ -direction. All DNS solutions sat-  
 101 isfy the standard convergence criteria (closed energy budget and fully resolved local Kol-  
 102 mogorov and Batchelor scales, Gayen *et al.* [2014]), which is tested for convergence across  
 103 a range of high- and low-resolution simulations. A no-slip condition is applied to the side  
 104 walls ( $y = 0, W$ ) and on the bottom topography of the channel. A rigid-lid condition is im-  
 105 posed at the top surface of the domain ( $z = H$ ), where a steady stress is applied for wind  
 106 forcing. The end conditions at  $x = 0$  and  $x = L$  are periodic in the zonal direction. All  
 107 boundaries are adiabatic except the top surface, where the temperature distribution is pre-  
 108 scribed and heating and cooling occurs. The system is considered to be in thermal equilib-  
 109 rium when the integrated flux over the heated area is within 5% of the integrated flux over  
 110 the cooled area, and fluctuations in kinetic energy are within 2% of the mean kinetic en-  
 111 ergy. All heat fluxes and mechanical energy conversions are calculated by time-averaging  
 112 over at least 45 buoyancy time scales,  $\tau_B$ , where  $\tau_B = \sqrt{W/g'}$ , and  $g'$  is reduced gravity,  
 113 defined as  $g' = g\alpha\Delta T$  based on the acceleration due to gravity,  $g$  and the co-efficient of  
 114 thermal expansion,  $\alpha$ .



**Figure 1.** An overview of the model domain, with overlaid flow solution for  $S = 4 \times 10^{-5}$ ,  $Ra = 1 \times 10^{12}$  and  $Ro = 4 \times 10^{-2}$ . *Blue line*: Imposed surface temperature; *White line*: prescribed wind stress. On the upper horizontal plane is shown the normalised instantaneous kinetic energy,  $\hat{E}_k$  inside the boundary layer at  $z/H = 0.991$ . The end view shows the normalised instantaneous zonally-averaged vertical velocity,  $\hat{w}$ , along with contours of normalised temperature,  $\hat{T}$  (white), corresponding to  $\hat{T} = 0.432, 0.200, 0.189, 0.187,$  and  $0.186$  (top to bottom contours, respectively). Velocities are normalised by the scale  $\sqrt{g'H}$  and temperature is expressed as  $(T - T_c)/\Delta T$ , where  $T_c$  is the coldest temperature.

## 2.2 Governing Parameters

The flow is governed by a number of non-dimensional parameters - the Rayleigh number ( $Ra$ ), Ekman number ( $E$ ), Prandtl number ( $Pr$ ), normalised beta-plane ( $\hat{\beta}$ ) and aspect ratios  $B$  and  $D$ :

$$Ra = \frac{g'W^3}{\nu\kappa}, E = \frac{\nu}{f_0W^2}, Pr = \frac{\nu}{\kappa}, \hat{\beta} = \frac{\beta W}{f_0}, B = \frac{L}{W}, D = \frac{H}{W}. \quad (1)$$

Here,  $\nu$  is kinematic viscosity and  $\kappa$  is thermal diffusivity. Molecular values of  $\nu$  and  $\kappa$  are used for all DNS simulations. A Rossby number  $Ro = U/f_0W$  can be defined by choosing any of several velocity scales. We use  $U \sim \sqrt{g'H}$ , so the Rossby number can be defined in terms of external parameters as  $Ro = \sqrt{g'H}/f_0W$ , leading to  $Ro = (RaE^2Pr^{-1}D)^{1/2}$ . In order to characterise the relative strength of wind- and buoyancy-forcing in this model, we introduce a parameter,  $S$ , which represents the ratio of the change in pressure due to imposed wind stress to that from a buoyancy differential:

$$S = \frac{\tau_{max}}{\rho_0g'H}, \quad (2)$$

based on reference density,  $\rho_0$ .

To compare the model results with the Southern Ocean, we determine the values of the relevant non-dimensional parameters ( $Ra$ ,  $Pr$ ,  $Ro$ ,  $E$  and  $S$ ) based on observational

129 values of the dimensional quantities in (1). Following *Hughes et al.* [2007] and *Vreug-*  
 130 *denhil et al.* [2016], we use estimates for turbulent viscosity and diffusivity,  $\nu_T = \kappa_T =$   
 131  $5 \times 10^{-5} - 2 \times 10^{-4} \text{ m}^2\text{s}^{-1}$  (50-200 times molecular values with  $Pr \approx 1$ ) to calculate  
 132 (1) for the Southern Ocean. The dimensions of the Southern Ocean are  $W = 3000 \text{ km}$ ,  
 133  $H = 5 \text{ km}$ , and  $L = 23000 \text{ km}$ . The maximum zonal wind stress is  $\tau_{max} \approx 0.2 \text{ Nm}^{-2}$   
 134 [*Large and Yeager, 2009*]. The reduced gravity for the system is  $g' = 0.017$ , with thermal  
 135 expansion co-efficient,  $\alpha$ , estimated at an average temperature and salinity of  $T = 6.7^\circ\text{C}$   
 136 and  $S \approx 34 \text{ psu}$  to be  $\alpha \sim 1 \times 10^{-4} \text{ C}^{-1}$  (from TEOS-10, [*McDougall et al., 2009*]) and  
 137  $\Delta T \approx 13.4^\circ\text{C}$  [*Beggs et al., 2011*]. Based on these values, the Rayleigh number,  $Ra$ , non-  
 138 dimensional parameter,  $S$ , Ekman number,  $E$ , and Rossby number,  $Ro$ , can be estimated  
 139 for the Southern Ocean, and are detailed in table 1.

140 Five numerical experiments with differing wind forcing ( $S$ ), buoyancy forcing ( $Ra$ ),  
 141 and rotation ( $E, Ro$ ) are conducted (table 1). In all cases,  $Pr = 5$ ,  $\hat{\beta} = 0.92$ ,  $D = 0.4$   
 142 and  $B = 5$ . The aspect ratio of the channel,  $D$ , is selected to ensure the thermal boundary  
 143 layer thickness,  $\delta_T$  is smaller than domain height ( $\delta_T \ll H$ ) for the range of  $Ra$  in this  
 144 study, where simulated  $\delta_T$  is calculated as the e-folding distance from the heated top sur-  
 145 face. We achieve  $O(10)$  Rossby deformation radii across the channel width, and the DNS  
 146 replicates a large ( $\sim 10^4$ ) scale separation between the turbulent Kolmogorov scale and the  
 147 basin scale (channel width). The simulations are conducted at the highest Rayleigh num-  
 148 ber possible given modern computational resources, but aspect ratio is greater than that  
 149 of the Southern Ocean, reducing the scale separation of the Rossby radius and convective  
 150 vortices.

Case	$Ra$	$S$	$E$	$Ro$
A	$1.125 \times 10^{12}$	0	$4 \times 10^{-7}$	0.12
B	$1.125 \times 10^{12}$	$4.44 \times 10^{-5}$	$4 \times 10^{-7}$	0.12
C	$1.125 \times 10^{12}$	$8.89 \times 10^{-5}$	$4 \times 10^{-7}$	0.12
D	$1.125 \times 10^{12}$	$1.77 \times 10^{-4}$	$4 \times 10^{-7}$	0.12
E	$3.75 \times 10^{10}$	$1.33 \times 10^{-3}$	$2 \times 10^{-6}$	0.11
SO	$1.2 \times 10^{25} - 1.9 \times 10^{26}$	$2.3 \times 10^{-6}$	$4.44 \times 10^{-14} - 1.78 \times 10^{-13}$	0.025

151 **Table 1.** The relevant non-dimensional parameters for the suite of simulations conducted (A-E). Case SO  
 152 represents the non-dimensional parameters for the Southern Ocean, calculated from observational values.

### 153 2.3 Energetic Framework

154 Local diapycnal mixing is:

$$\Phi_d = -\kappa g \left( \frac{\partial z^*}{\partial \rho} \right) \left( \frac{\partial \rho}{\partial x_j} \right)^2, \quad (3)$$

155 where  $z^*$  is the height above the base at which each water parcel would be found when  
 156 the fluid was adiabatically resorted to a statically stable state, as done here [*Winters et al.,*  
 157 1995]. Local viscous dissipation is defined as the spatial gradient of the total instanta-  
 158 neous velocity:

$$\varepsilon = \rho_0 \nu (\partial u_i / \partial x_j)^2. \quad (4)$$

159 The individual terms in the energy budget are normalised by the domain averaged  
 160 rate of potential energy generation from internal energy:

$$\Phi_i = \kappa g A \bar{\rho}_{top} - \kappa g \int \int \rho(x, y, h(x, y)) dx dy \quad (5)$$

161 where  $A$  is the area of the top surface of the domain,  $\bar{\rho}_{top}$  is the average density at the  
 162 top surface and  $\rho(x, y, h(x, y))$  is the bottom density along the topography. The average  
 163 mixing efficiency is defined as the ratio of diapycnal mixing to the total mechanical en-  
 164 ergy sink, expressed mathematically as [Peltier and Caulfield, 2003]:

$$\eta = \frac{\langle \Phi_d \rangle - \langle \Phi_i \rangle}{\langle \Phi_d \rangle - \langle \Phi_i \rangle + \langle \varepsilon \rangle}, \quad (6)$$

165 where the angled brackets indicate a volume integral. While other definitions of mixing  
 166 efficiency exist (see Osborn [1980]; Ivey and Imberger [1991]), we adopt (6) because it  
 167 is unaffected by the large reversible energy fluxes in buoyancy-dominated flows such as  
 168 convection.

169 We also define a local mixing efficiency,  $\eta_L = \Phi_d / (\Phi_d + \varepsilon)$ ; this definition differs  
 170 from (6) as it omits  $\Phi_i$  to ensure that  $\eta_L$  cannot exceed 1. The terms in (6) are volume-  
 171 integrated, hence the volume-averaged  $\eta_L$  will not equal to average mixing efficiency,  $\eta$ ,  
 172 i.e.,  $\frac{1}{V} \int \eta_L dV \neq \eta$ .

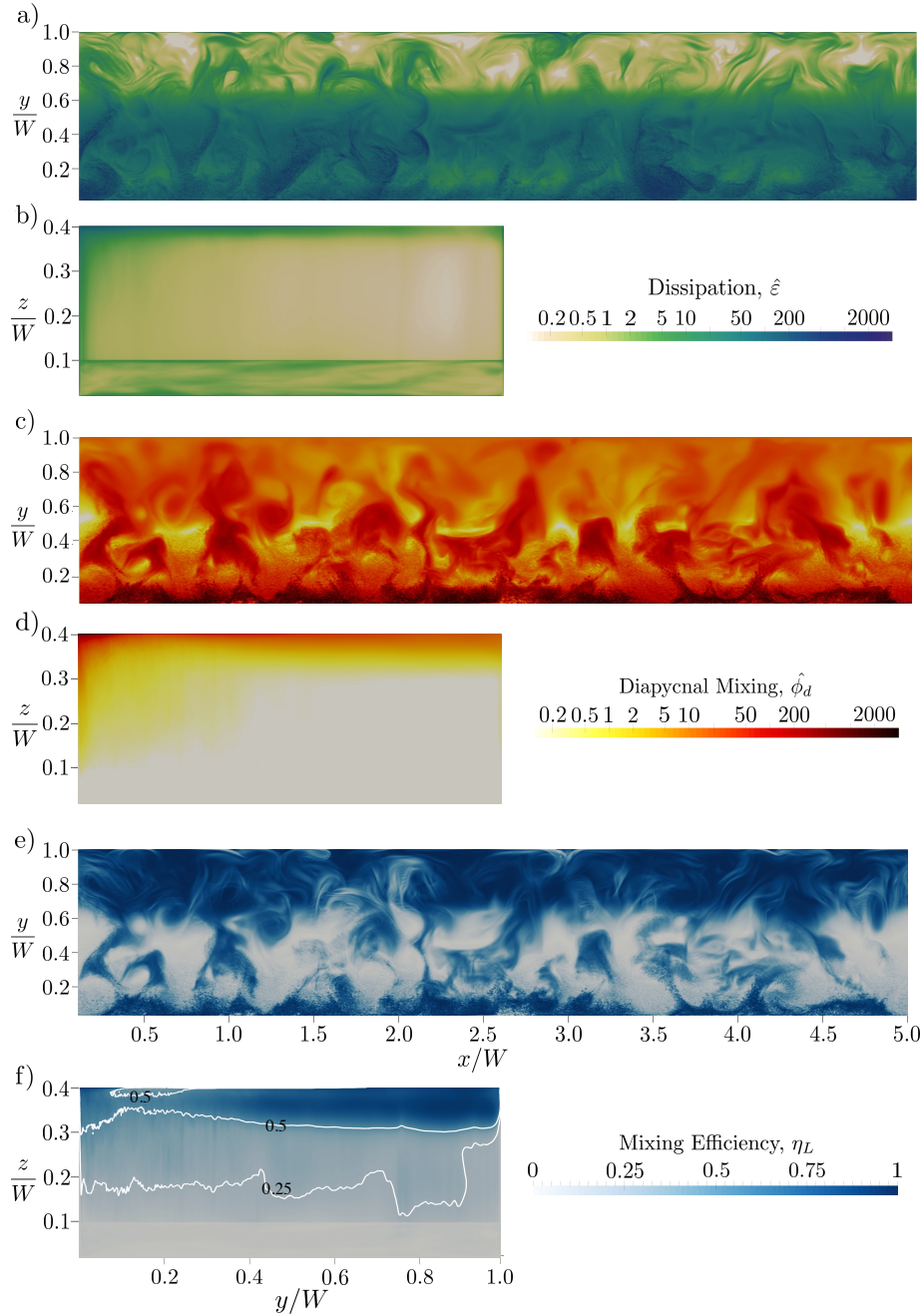
### 173 3 Results

174 The thermally equilibrated model exhibits a wide range of scales of motion, from  
 175 small-scale convection to geostrophic flow (figure 1) with a vertical stratification (white  
 176 lines show isopycnals on the  $y - z$  plane in figure 1). Wind stress and surface buoy-  
 177 ancy fluxes combine to steepen the isopycnals, which excites baroclinic eddies, and sub-  
 178 sequently flattens the isopycnals [Karsten *et al.*, 2002]. These eddies can be seen in the  
 179 kinetic energy plot on the top surface of the domain (figure 1), and are also present in the  
 180 zonally-averaged view as regions of transient upwelling and downwelling (see online mul-  
 181 timedia). The stratification also results in the formation of a braided, meandering zonal jet  
 182 along the channel through thermal wind balance [Hogg, 2010]. Small-scale vertical con-  
 183 vection is present in the region of destabilising buoyancy, with a persistent downwelling  
 184 plume present along the southern boundary (vertical velocity in figure 1). Thus, the model  
 185 exhibits many of the observed characteristics of flow in the Southern Ocean.

191 There is spatial variation of both local dissipation and diapycnal mixing (figure 2a-  
 192 d). Largest dissipation rates are observed inside the top boundary layer and are concen-  
 193 trated in the horizontal band of applied wind stress which spans 3/4 of the channel width  
 194 (figure 2a,b). The fronts associated with zonal jets and baroclinic eddies are regions of  
 195 high dissipation [Barkan *et al.*, 2015]. Dissipation is also large in the turbulent convec-  
 196 tive plume. Dissipation is low in the interior away from the convective zone and the stably  
 197 stratified boundary layer. Diapycnal mixing is similarly concentrated in regions of active  
 198 convection and inside the top boundary layer (figure 2c, d), as a result of the large near-  
 199 surface stratification, which is enhanced due to shear associated with the mean flow.

200 Local mixing efficiency,  $\eta_L$ , varies significantly over the domain (figure 2e-f).  $\eta_L$   
 201 is high where convection is dominant and in the stably stratified boundary layer region,  
 202 where diapycnal mixing is large. Dissipation is enhanced by surface wind stress, lead-  
 203 ing to smaller local mixing efficiency in the thin boundary layer underlying the horizontal  
 204 band of wind stress (not visible in the figure). The variation of both diapycnal mixing and  
 205 dissipation rate with depth is qualitatively similar. However, diapycnal mixing is smaller  
 206 than dissipation throughout the interior, giving  $\eta_L < 0.5$  in the interior (zonally averaged).  
 207 Mixing is greater than dissipation in the upper 25% of the domain, giving  $\eta_L > 0.5$ , and  
 208  $\eta_L$  approaches 1 in the northern 30% of the domain.

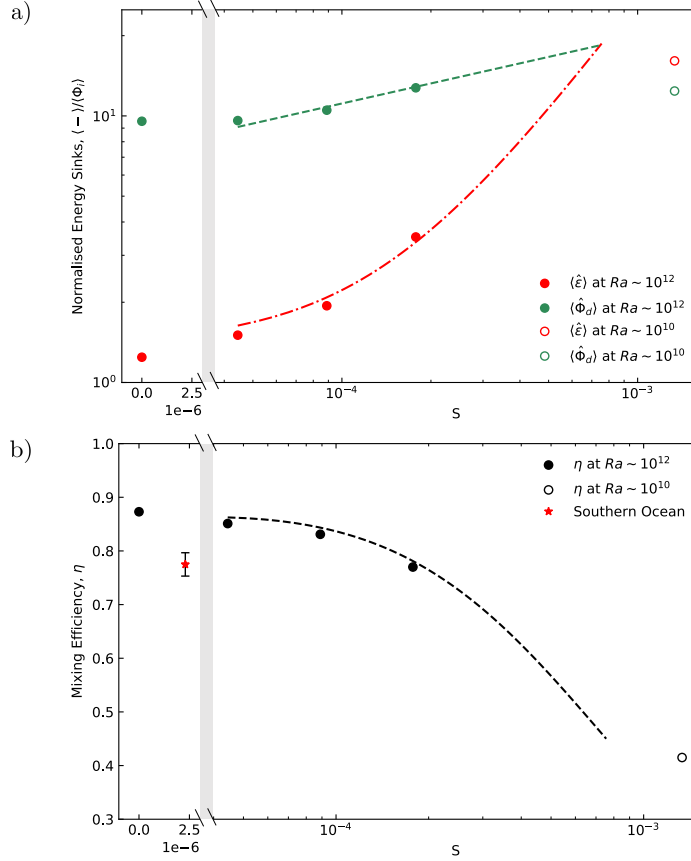
209 Figure 3a shows the change in normalised, time-averaged and volume-integrated di-  
 210 apycnal mixing,  $\langle \Phi_d \rangle$ , and viscous dissipation rate,  $\langle \varepsilon \rangle$  with wind stress. Dissipation in-  
 211 creases with increased wind stress because the majority of momentum imparted by the  
 212 wind stress is dissipated very close to the surface of the model. At small wind stress the



186 **Figure 2.** Time-averaged spatial distribution of energetics terms for case D: a) top surface and b) zonally-  
 187 averaged dissipation,  $\hat{\varepsilon}$ ; c) top surface and d) zonally-averaged diapycnal mixing,  $\hat{\phi}_d$ ; and e) top surface and f)  
 188 zonally-averaged local mixing efficiency,  $\eta_L$ . The top surface plots are at a depth of  $z/W = 0.398$ , within the  
 189 thermal boundary layer. Energetics are normalised by  $\Phi_i$ . Contour lines in (f) show mixing efficiency of 0.5  
 190 and 0.25.

213 dissipation approaches the theoretical limit,  $\langle \varepsilon \rangle \rightarrow \langle \Phi_i \rangle$  (represented by case A) as noted  
 214 by *Paparella and Young* [2002]. Diapycnal mixing also increases with increasing wind  
 215 stress. Increasing the imposed wind stress steepens the isopycnals, thereby increasing the  
 216 surface buoyancy uptake and consequently increasing diapycnal mixing (which is depen-  
 217 dent only on buoyancy uptake, *Hughes et al.* [2013]). This indirect coupling between wind

218 stress and mixing is highlighted by ocean model results [Saenz *et al.*, 2012]. The absolute  
 219 value of diapycnal mixing is greater than the changes of mixing with wind stress, imply-  
 220 ing that buoyancy forcing is the dominant driver of diapycnal mixing within the range of  
 221 conditions studied here.



222 **Figure 3.** a) The time-averaged, volume-integrated normalised dissipation,  $\langle \hat{\epsilon} \rangle$  and diapycnal mixing,  $\langle \widehat{\Phi}_d \rangle$   
 223 as functions of the relative wind stress,  $S$ , and b) variation of mixing efficiency,  $\eta$ , with the relative wind stress  
 224  $S$ . Filled Circles: cases A - D; Open Circle: case E. The energetics are normalised by  $\langle \Phi_i \rangle$ . Lines are the  
 225 predictions of the scaling theory.

226 Mixing efficiency, calculated from equation (6), is shown in figure 3b as a func-  
 227 tion of  $S$ . The mixing efficiency is large,  $0.8 < \eta < 0.9$ , in cases A - D. Case E (open  
 228 circle) lies in the laminar boundary layer regime (identified by Gayen *et al.* [2013b] as  
 229  $Ra < 10^{11}$ ), and consequently has a reduced mixing efficiency (consistent with Barkan  
 230 *et al.* [2013]). This result suggests that turbulent convection enhances diapycnal mixing  
 231 and mixing efficiency. At small wind stress mixing efficiency is asymptotic to the theo-  
 232 retical value given by (6) with  $\langle \epsilon \rangle = \langle \Phi_i \rangle$ , so  $\eta = 1 - \langle \Phi_i \rangle / \langle \Phi_d \rangle$  as shown in previous  
 233 horizontal convection studies [Vreugdenhil *et al.*, 2016]. When mixing efficiency is calcu-  
 234 lated with turbulent, instead of total, dissipation in (6), it does not vary significantly.

#### 235 4 Application to Ocean

236 To apply the model results to the Southern Ocean (where Rayleigh number is higher  
 237 and aspect ratio smaller), scaling theories are needed. In thermal equilibrium the diapyc-

238 nal mixing rate is given by  $\langle \Phi_d \rangle = \langle \Phi_{b2} \rangle$  [Hughes *et al.*, 2009], where  $\langle \Phi_{b2} \rangle$  is the me-  
239 chanical power input by surface buoyancy fluxes, defined as:

$$\langle \Phi_{b2} \rangle = -\kappa g \oint z^* (\partial \rho / \partial x_i) dA, \quad (7)$$

240 where  $A$  is the top surface of the domain. Viscous dissipation equates to the sum of the  
241 background rate of molecular diffusion and the kinetic energy input by surface wind stress  
242 into the system:

$$\langle \varepsilon \rangle = \langle \Phi_\tau \rangle + \langle \Phi_i \rangle, \quad (8)$$

243 where  $\langle \Phi_\tau \rangle$  is wind work, expressed as:

$$\langle \Phi_\tau \rangle = \oint \tau u|_{z=H} dA, \quad (9)$$

244 and  $u|_{z=H}$  is velocity at the top surface. Therefore, if the power inputs by buoyancy and  
245 wind work are known, the diapycnal mixing and viscous dissipation rates can be pre-  
246 dicted.

247 In thermal equilibrium the net heat flux through every horizontal level of the do-  
248 main is zero. Therefore, the buoyancy uptake can be estimated from the thickness,  $\delta$ , of  
249 the stably stratified region of the thermal boundary layer. With surface buoyancy forc-  
250 ing alone a horizontal thermal wind balance and vertical advection-diffusion balance is  
251 established and the thermal boundary layer thickness can consequently be expressed as  
252  $\delta \sim (\kappa f_0 W^2 / g')^{1/3}$  [Robinson and Stommel, 1959; Welander, 1971; Winton, 1996; Park  
253 and Bryan, 2000; Vreugdenhil *et al.*, 2016]. When wind stress is introduced, the verti-  
254 cal advection near the surface is dominated by Ekman pumping with vertical velocity  
255  $w_E \sim \tau_{max} / (\rho_0 f_0 W)$ , resulting in a "modified" thermal boundary layer thickness of  $\delta_T \sim$   
256  $[\rho_0 f_0^2 \kappa^2 W^3 / (g' \tau_{max})]^{1/4}$  [see derivation in Samelson and Vallis, 1997].  $\delta_T$  can be ex-  
257 pressed in terms of non-dimensional parameters as:

$$\delta_T / W = c_{\delta_T} (S Ro^2 Ra Pr)^{-1/4}, \quad (10)$$

258 where the value of the prefactor  $c_{\delta_T} = 0.70$  can be determined from simulations.

259 The power input by buoyancy,  $\langle \Phi_{b2} \rangle$ , is defined in terms of the resorted height,  $z^*$ .  
260 We find that  $z^* \approx H$  in the heated region of the domain (where less dense water parcels  
261 exist) and  $z^* \approx 0$  in the cooled section (where more dense water parcels are present).

262 Hence, the diapycnal mixing can be approximated by  $\langle \Phi_d \rangle = \langle \Phi_{b2} \rangle \approx \kappa \rho_0 g \alpha a_h \overline{dT/dz} \Big|_h \sim$   
263  $\kappa \rho_0 g \alpha a_h H (\Delta T / \delta_T)$ , where the subscript  $h$  indicates the heated region and  $\overline{dT/dz} \Big|_h$  is  
264 the spatially averaged vertical temperature gradient at the boundary in the heated region,  
265 which has area  $a_h$ . The normalised diapycnal mixing is then:

$$\langle \widehat{\Phi}_d \rangle = \frac{\langle \Phi_d \rangle}{\langle \Phi_i \rangle} \sim \frac{H}{\delta_T} \sim (S Ro^2 Pr Ra D^4)^{1/4}, \quad (11)$$

266 where the rate of generation of potential energy is  $\langle \Phi_i \rangle \approx \kappa \rho_0 g \alpha \Delta T a_h$ .

267 To predict viscous dissipation, we first estimate the wind work, which scales as  
268  $\langle \Phi_\tau \rangle \sim u_a \tau_{max} W L$ . We assume that ageostrophic motion dominates surface zonal flow  
269 when wind forcing is sufficiently high. Therefore, the velocity in (9) is  $u_a$ , the ageostrophic  
270 zonal velocity scale. From Ekman layer theory [Ekman, 1905], the ageostrophic surface  
271 zonal flow is  $u_a \sim \tau_{max} / (\rho_0 f_0 \delta_E)$ , where  $\delta_E$  is the turbulent Ekman layer thickness,  
272  $\delta_E = 0.5 \sqrt{\tau_{max} / \rho_0} / f_0$  [Coleman *et al.*, 1990; Price and Sundermeyer, 1999; Wang and  
273 Huang, 2004]. The normalised wind work becomes:

$$\langle \widehat{\Phi}_\tau \rangle = \frac{\langle \Phi_\tau \rangle}{\langle \Phi_i \rangle} \sim (S^3 Ra Pr D^3)^{1/2}. \quad (12)$$

274 From (8), the normalised viscous dissipation rate at thermal equilibrium is estimated as:

$$\langle \widehat{\varepsilon} \rangle = \frac{\langle \varepsilon \rangle}{\langle \Phi_i \rangle} \sim 1 + (S^3 Ra Pr D^3)^{1/2}. \quad (13)$$

275 The scaling for diapycnal mixing (11) and dissipation (13) is tested against the com-  
 276 puted flow solutions in figure 3a. Both quantities follow the scaling predictions closely.  
 277 The average mixing efficiency predicted by (6) is shown in figure 3b and the theoretical  
 278 estimation is in good agreement with the DNS. However, the scaling theory does not pro-  
 279 vide a good prediction for case E because this simulation lies outside the turbulent con-  
 280 vective regime ( $Ra > 10^{11}$ ; *Gayen et al.* [2014]).

281 From the scaling and ocean variables given in section 2.2, we determine that the  
 282 thermal boundary layer thickness for the Southern Ocean is  $\delta_T \approx [200 - 400]$  m. The  
 283 rate of conversion from internal to potential energy is estimated to lie in the range  $\langle \Phi_i \rangle =$   
 284  $[0.04 - 0.2]$  TW, with  $a_h$  assumed to be 70% of the surface area of the Southern Ocean.  
 285 The scaling gives a diapycnal mixing rate  $\langle \Phi_d \rangle \approx [0.9 - 2]$  TW. The wind work is pre-  
 286 dicted to be  $\langle \Phi_\tau \rangle \approx 0.22$  TW, which is close to inferences from observations [*Wunsch,*  
 287 *1998; Hughes and Wilson, 2008; Roquet et al., 2011*]. From (13), we estimate that dis-  
 288 sipation is  $\langle \varepsilon \rangle \approx [0.3 - 0.4]$  TW. From (6) we predict an average mixing efficiency  
 289  $\eta \approx [0.75 - 0.80]$  for the Southern Ocean.

## 290 5 Discussion & Conclusions

291 Energetically closed, turbulence-resolving solutions with wind- and buoyancy-forcing  
 292 are used to investigate the impact of convection on diapycnal mixing and turbulent dissipa-  
 293 tion in our model of the Southern Ocean. Local dissipation is highest near the surface of  
 294 the domain underlying wind stress forcing, and in the regions of deep convection, whereas  
 295 local diapycnal mixing is highest in regions of turbulent convection, as well as in the stably  
 296 stratified region in the northern half of the domain. Therefore, highest values of local  
 297 mixing efficiency are observed inside the deep convection region and in the buoyancy sta-  
 298 bilising region, where a strong vertical buoyancy gradient is present. Overall, small-scale  
 299 convection enhances volume-integrated diapycnal mixing to be greater than dissipation so  
 300 that mixing efficiency is in the range  $\eta \approx 0.8 - 0.9$  within oceanographically realistic  
 301 values of  $S$ . Increasing wind stress enhances both volume-integrated dissipation and di-  
 302 apycnal mixing, but ultimately reduces mixing efficiency. Using a scaling analysis which  
 303 matches the DNS results, we predict that mixing efficiency in the Southern Ocean might  
 304 have a value of as high as  $\eta \approx 0.78$ . These findings indicate that convection increases  
 305 mixing efficiency and the results cast doubt on assumptions that mixing efficiency is uni-  
 306 form and is controlled solely by shear-driven instabilities. In addition, the spatial variance  
 307 of mixing efficiency indicates regions where the majority of energy loss goes to mixing.  
 308 This information may be used in targeted observational studies of mixing in the future.

309 We have focussed here on the role of wind and buoyancy forcing on dissipation and  
 310 diapycnal mixing. However, tides have been predicted to introduce up to 3.7 TW of me-  
 311 chanical energy into the global ocean. Approximately 0.6 – 0.9 TW of this tidal energy  
 312 may contribute to diapycnal mixing [*Munk and Wunsch, 1998*]. Further studies with a  
 313 turbulence-resolving model incorporating tidal mixing would be useful in elucidating the  
 314 impact of tidal forcing on the mechanical energy budget of the Southern Ocean. There is  
 315 also a significant difference between the aspect ratios used in the present model and those  
 316 in the ocean. While this difference is taken into account in the theoretical scaling analysis  
 317 used here to predict the energetics of the Southern Ocean, the accuracy of the aspect ratio  
 318 dependence has not been tested in the DNS.

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Figure 1.

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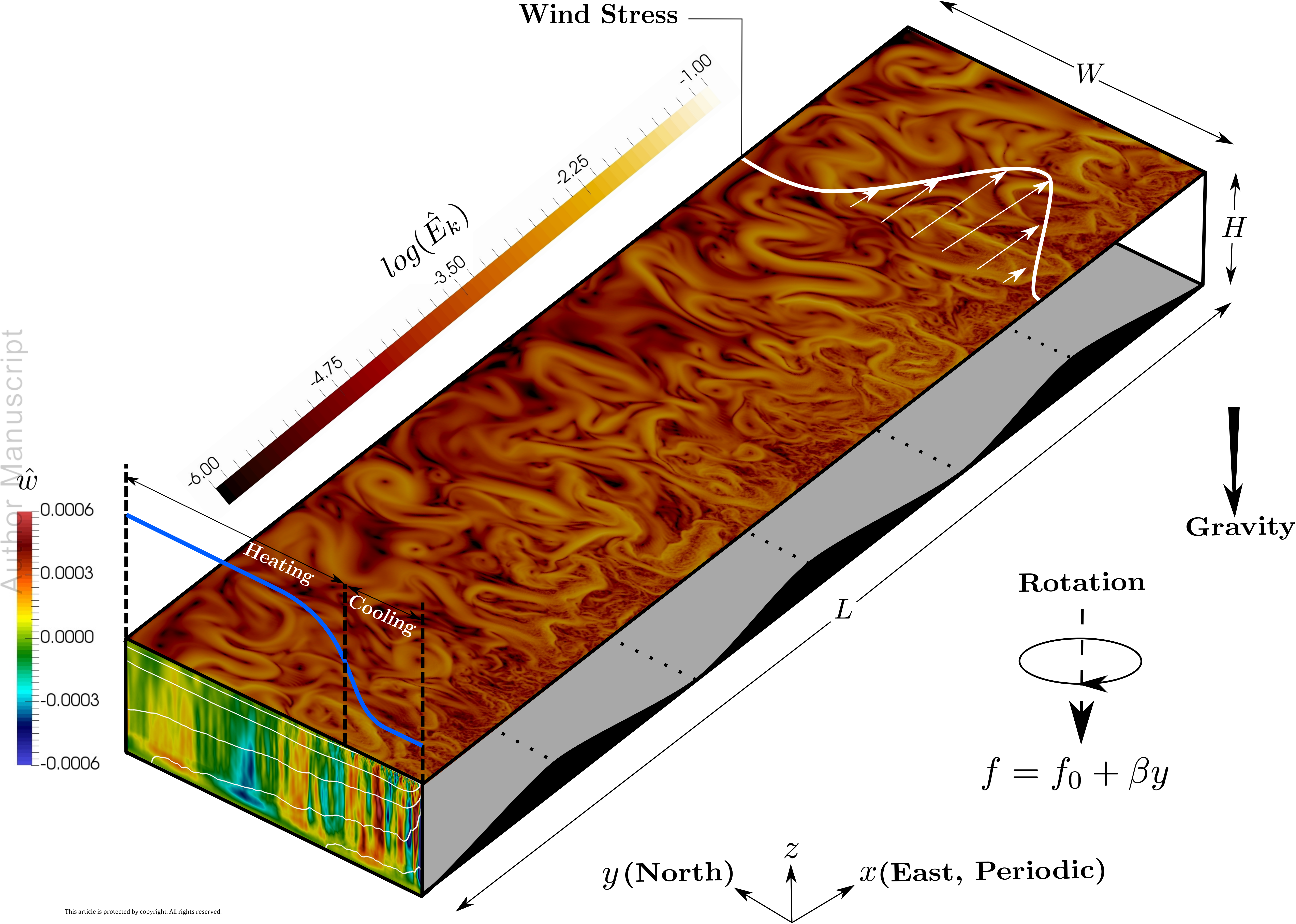


Figure 2.

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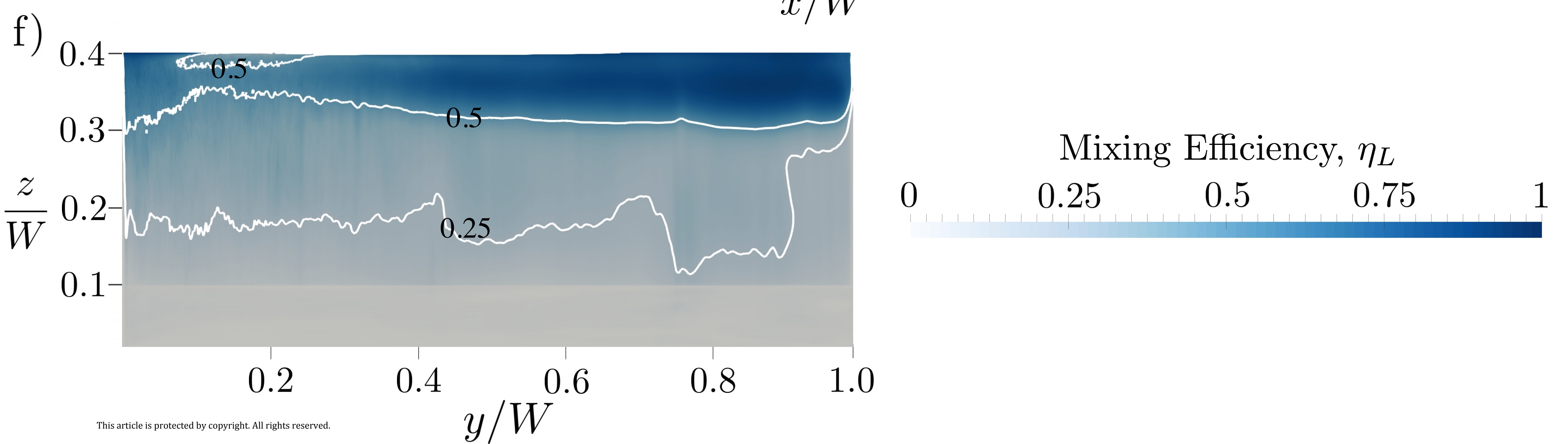
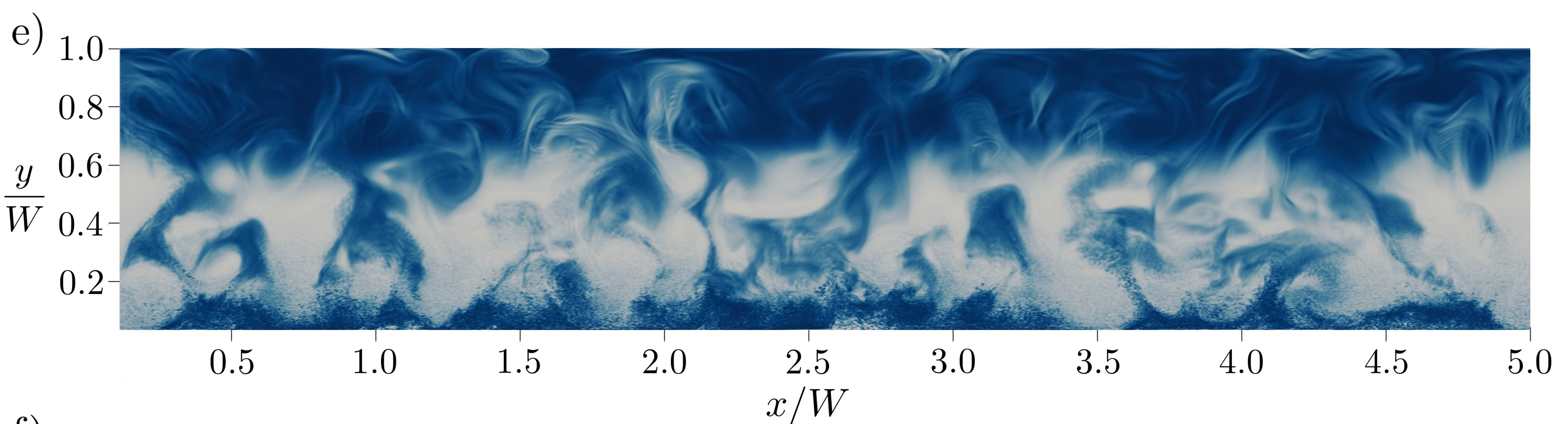
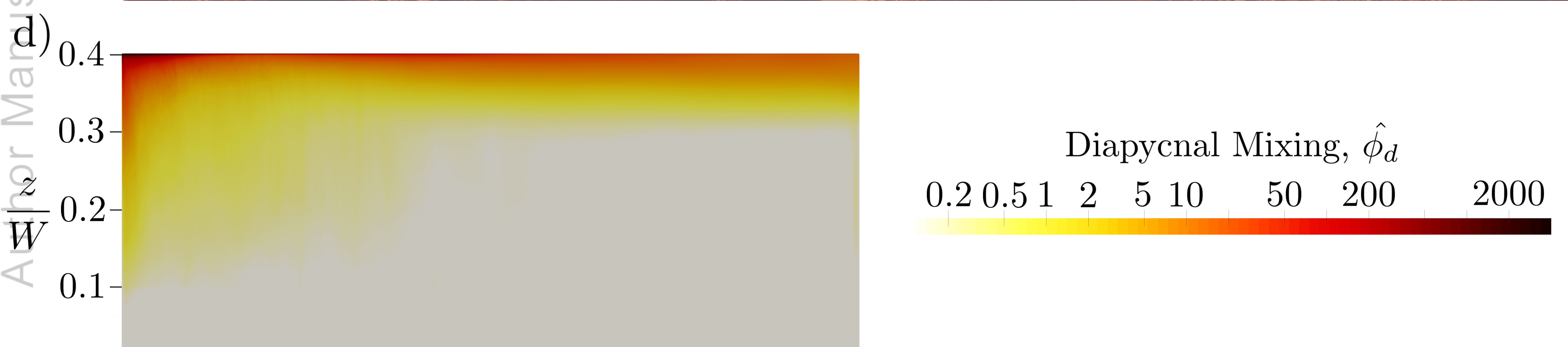
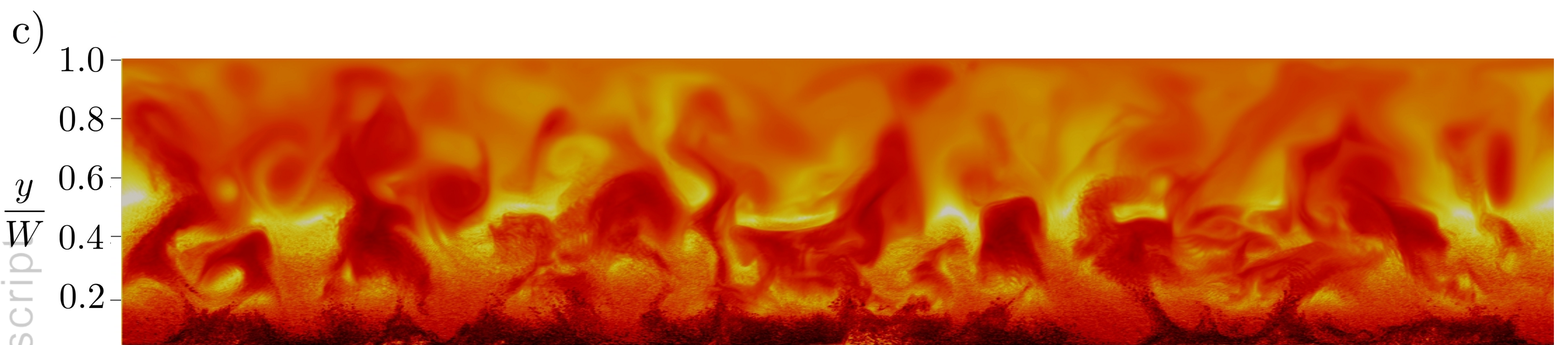
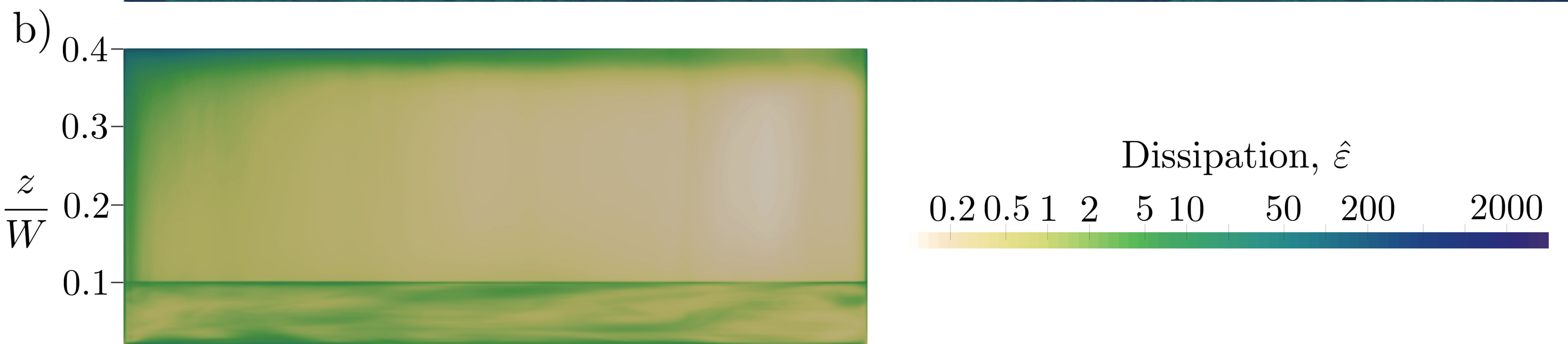
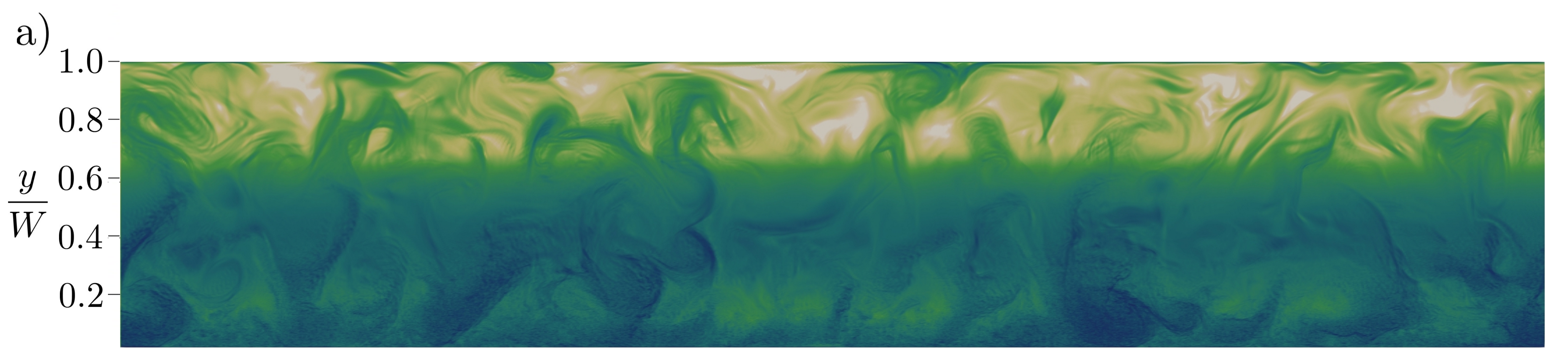
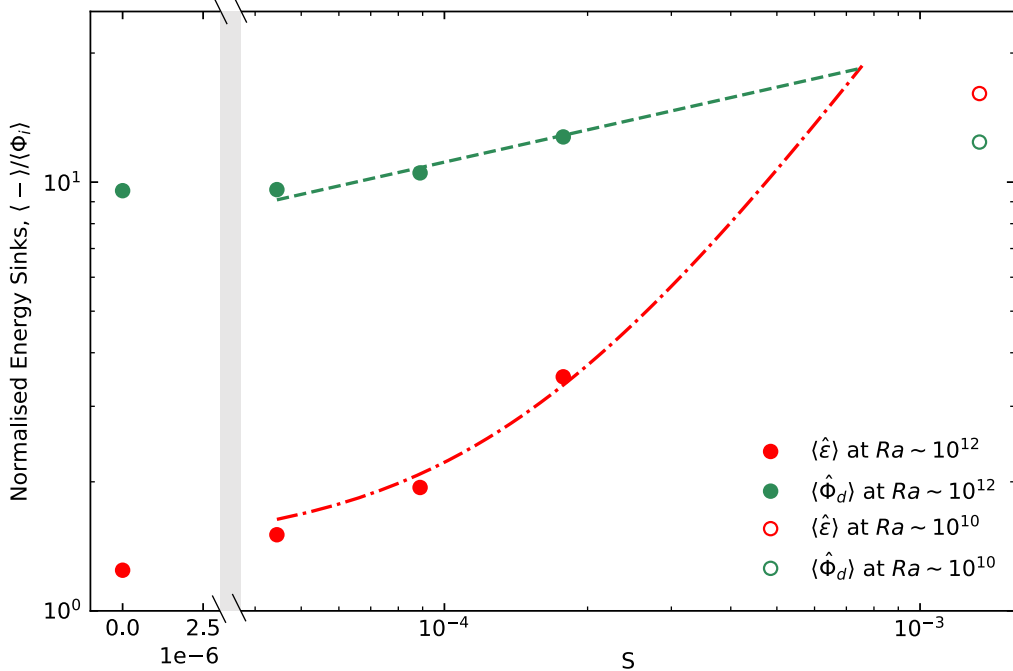


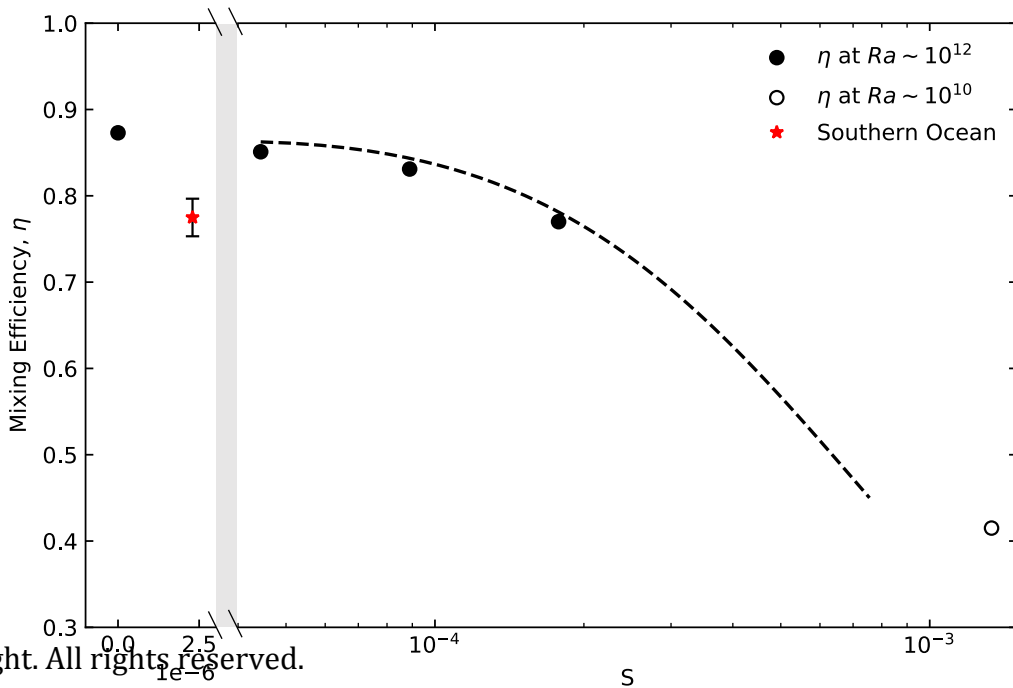
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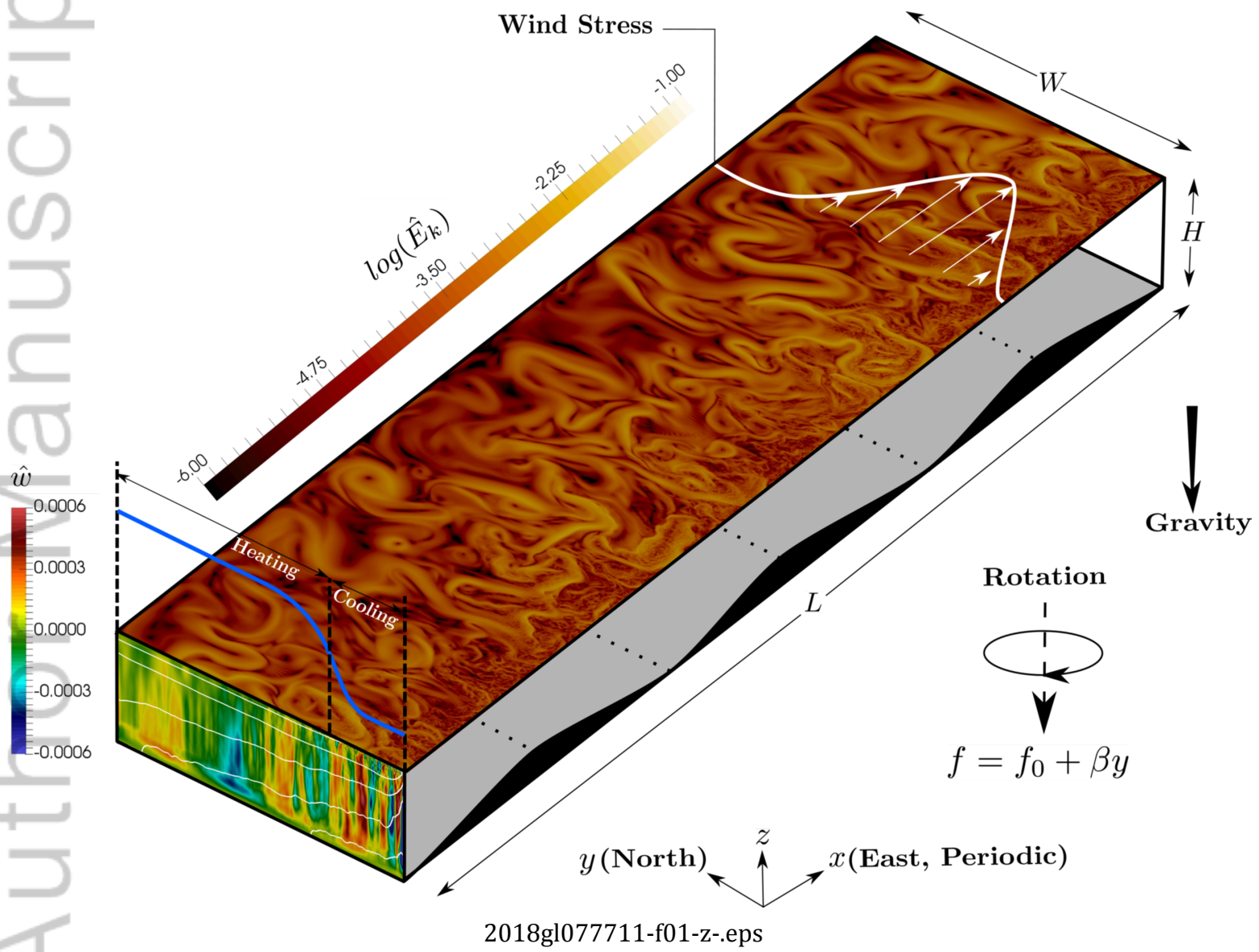
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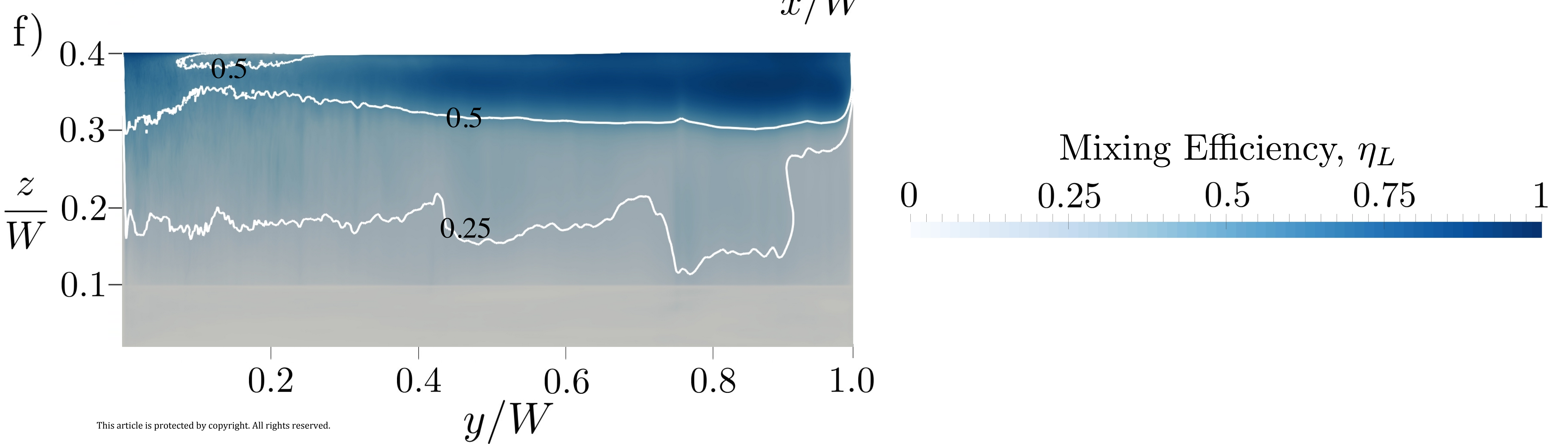
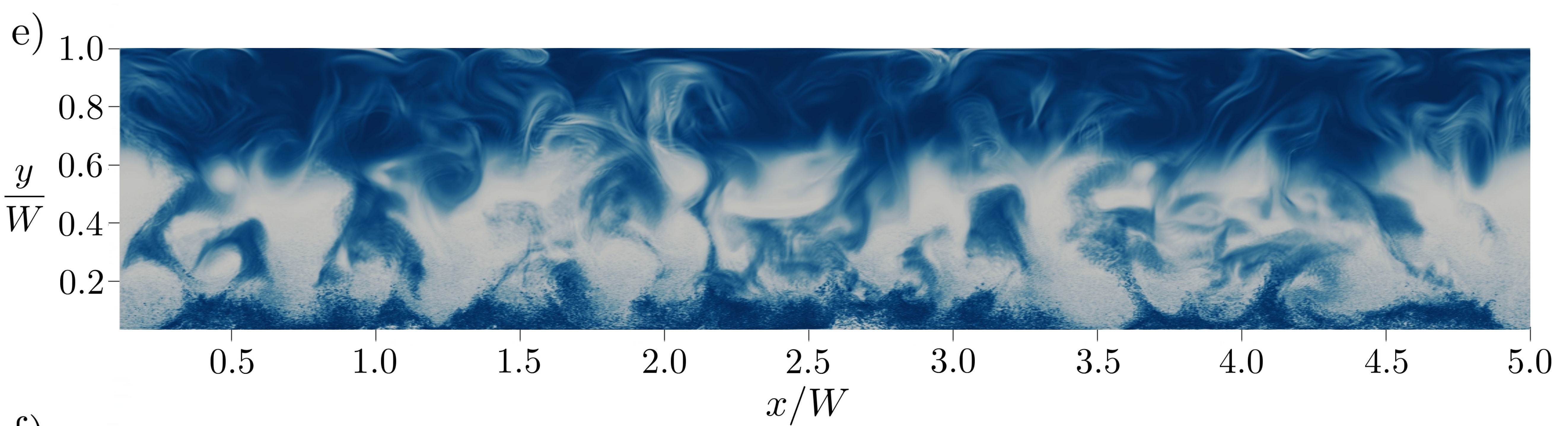
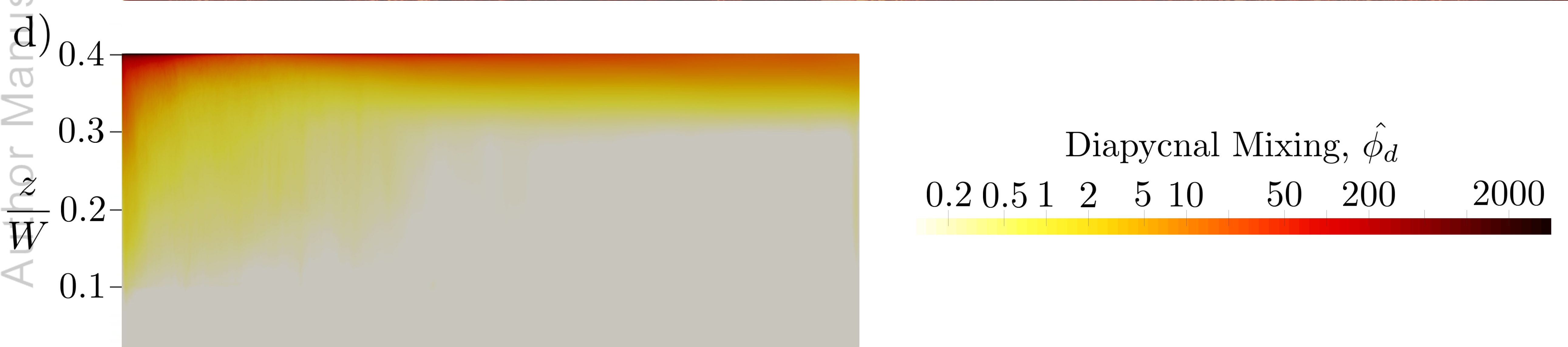
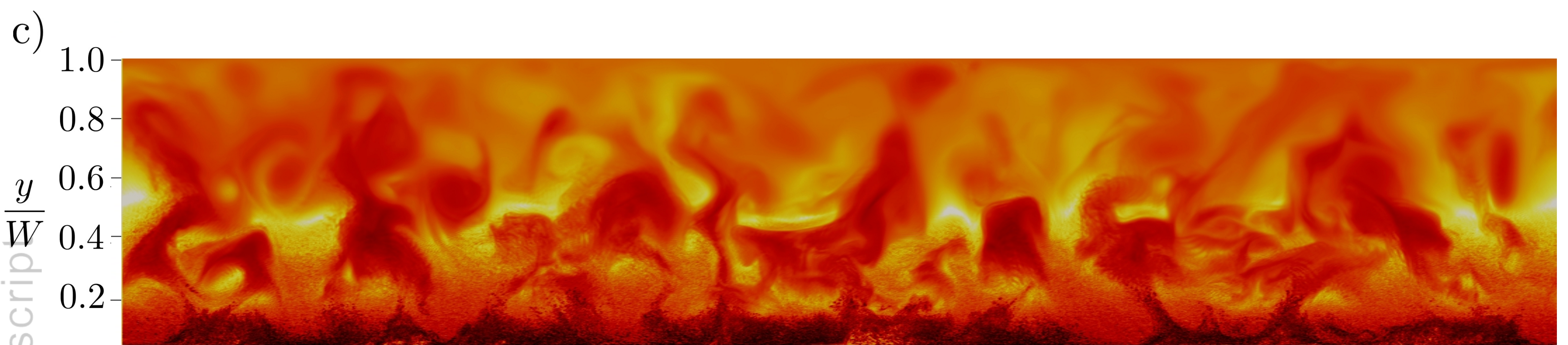
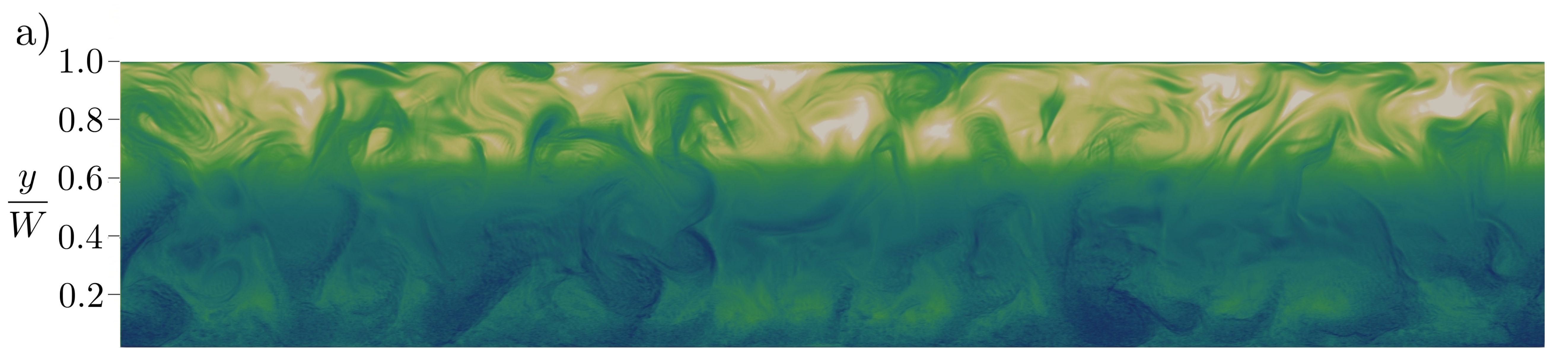


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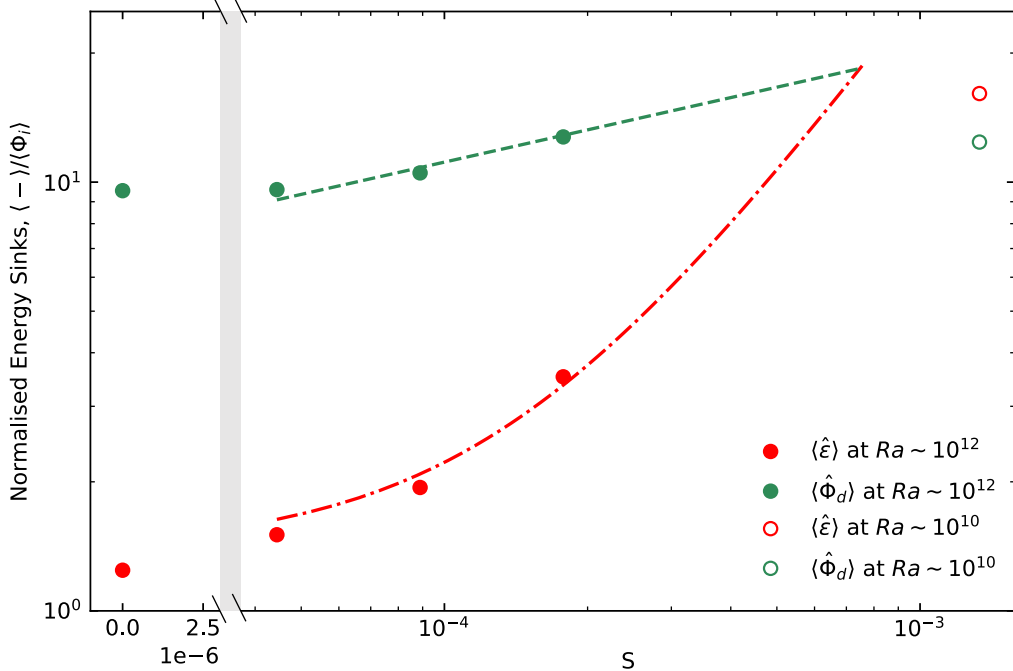




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b)

